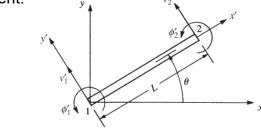
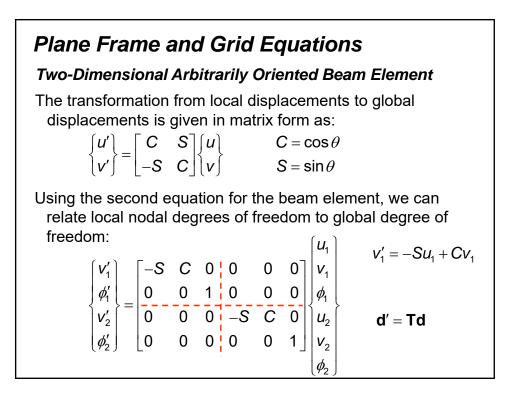


Two-Dimensional Arbitrarily Oriented Beam Element

We can derive the stiffness matrix for an arbitrarily oriented beam element, in a manner similar to that used for the bar element. v_2



The local axes and are located along the beam element and transverse to the beam element, respectively, and the global axes x' and y' are located to be convenient for the total structure.

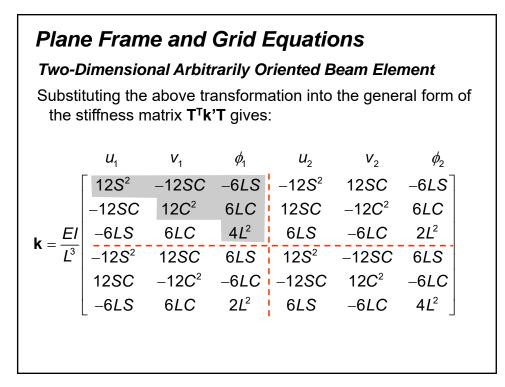


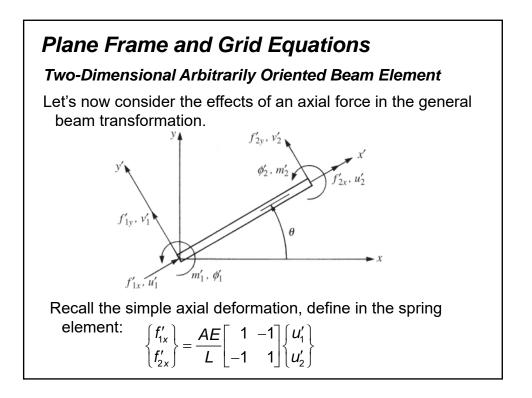
Two-Dimensional Arbitrarily Oriented Beam Element

For a beam, we will define the following as the *transformation matrix*:

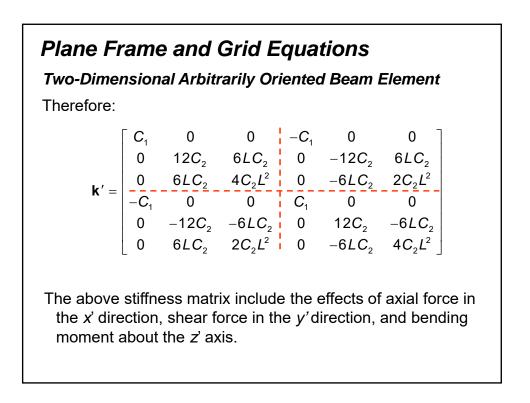
$$\mathbf{T} = \begin{bmatrix} -S & C & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & -S & C & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}$$

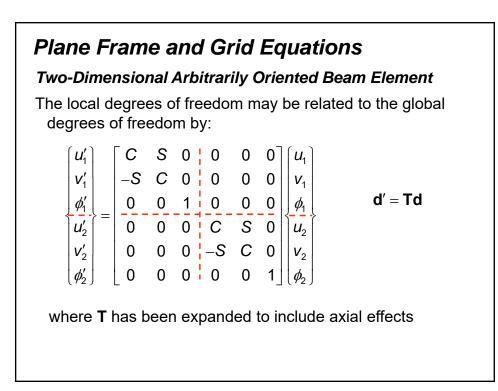
Notice that the rotations are not affected by the orientation of the beam.





$\begin{array}{l} \textbf{Plane Frame and Grid Equations}\\ \textbf{Two-Dimensional Arbitrarily Oriented Beam Element}\\ \textbf{Combining the axial effects with the shear force and bending moment effects, in local coordinates gives:}\\ \begin{pmatrix} f'_{1x} \\ f'_{2y} \\ m'_{1} \\ f'_{2y} \\ m'_{2} \end{pmatrix} = \begin{pmatrix} C_1 & 0 & 0 & -C_1 & 0 & 0 \\ 0 & 12C_2 & 6LC_2 & 0 & -12C_2 & 6LC_2 \\ 0 & 6LC_2 & 4C_2L^2 & 0 & -6LC_2 & 2C_2L^2 \\ -C_1 & 0 & 0 & C_1 & 0 & 0 \\ 0 & -12C_2 & -6LC_2 & 0 & 12C_2 & -6LC_2 \\ 0 & 6LC_2 & 2C_2L^2 & 0 & -6LC_2 & 4C_2L^2 \\ \end{pmatrix} \begin{pmatrix} u'_1 \\ v'_1 \\ u'_2 \\ v'_2 \\ v'_2 \\ v'_2 \end{pmatrix} \\ \mathcal{L}_1 = \frac{AE}{L} \qquad C_2 = \frac{EI}{L^3} \end{array}$

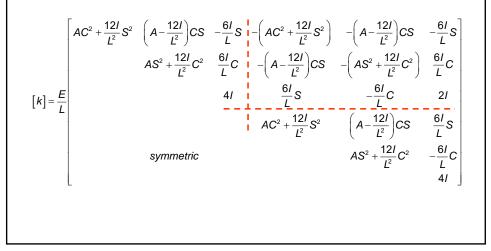


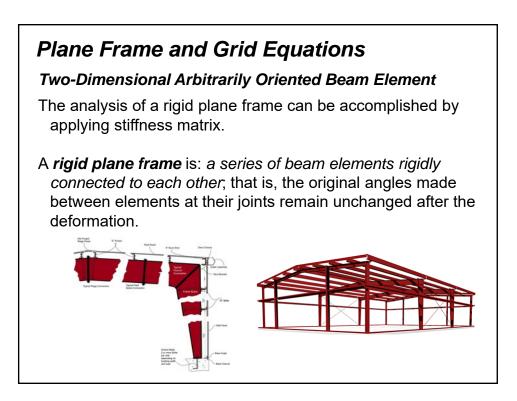


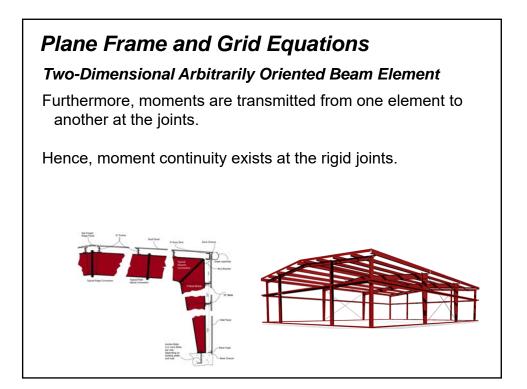


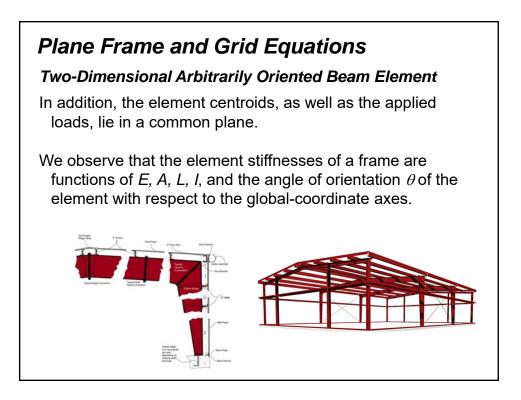
Two-Dimensional Arbitrarily Oriented Beam Element

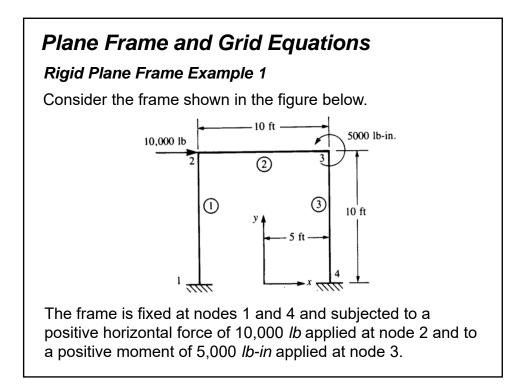
Substituting the above transformation **T** into the general form of the stiffness matrix gives:

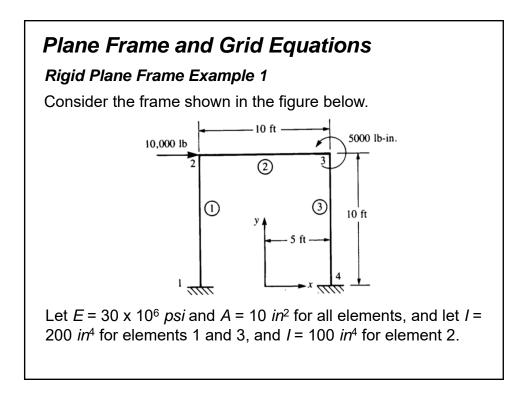


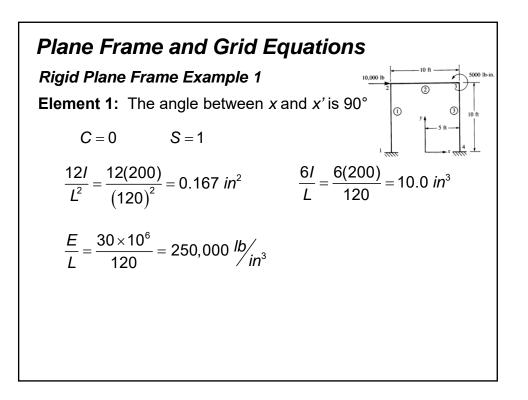


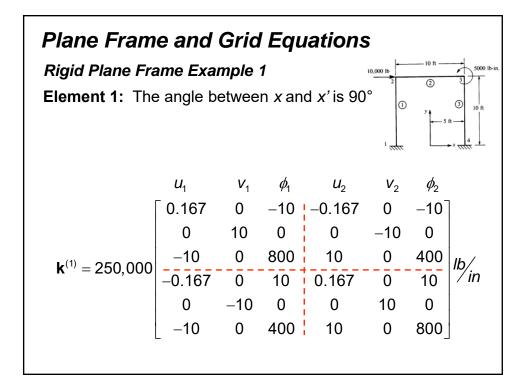


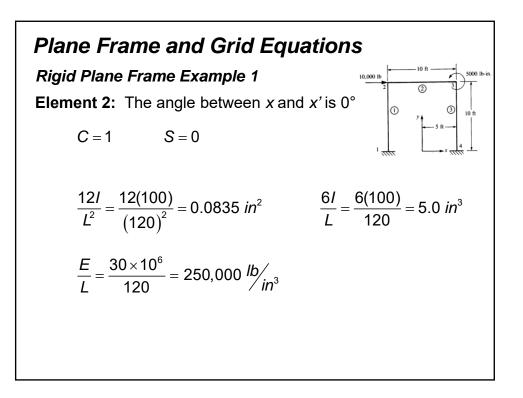


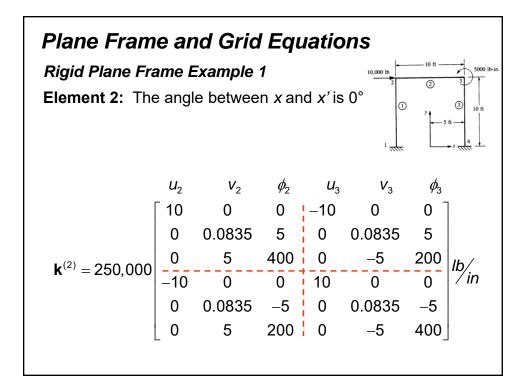


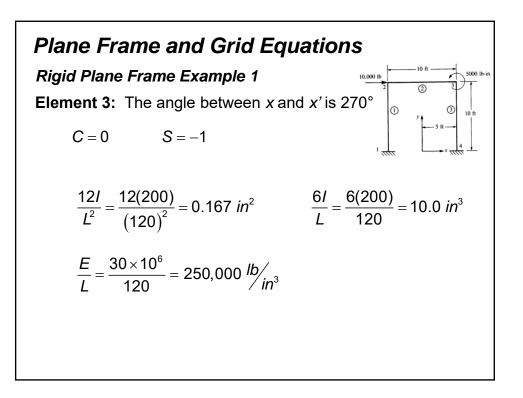


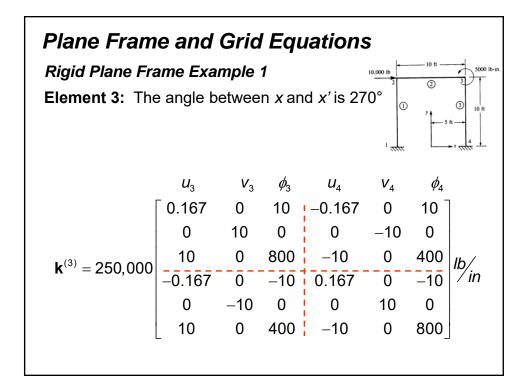


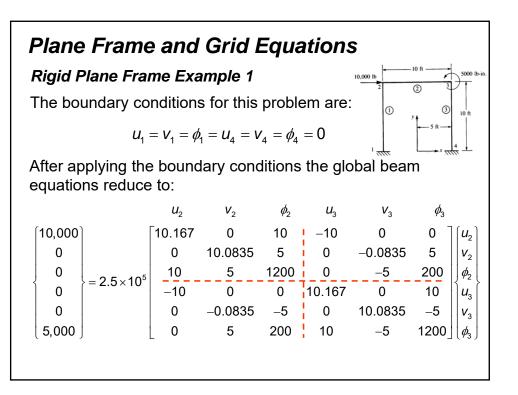


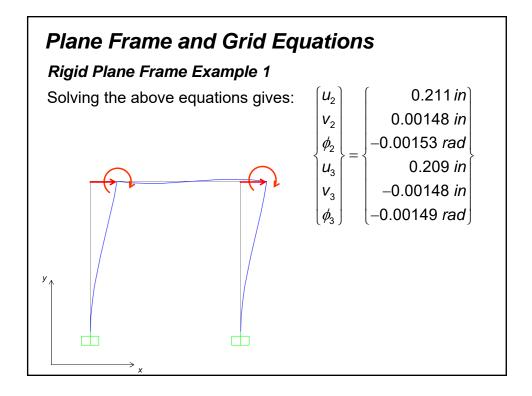


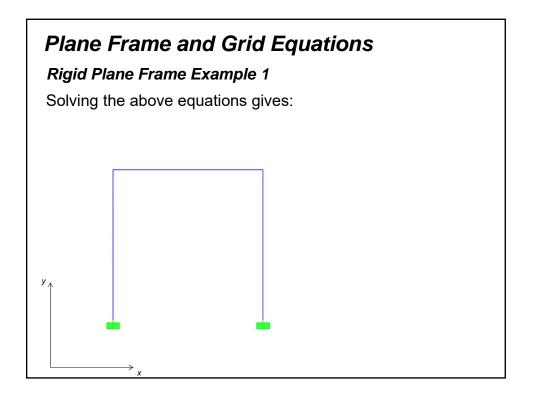




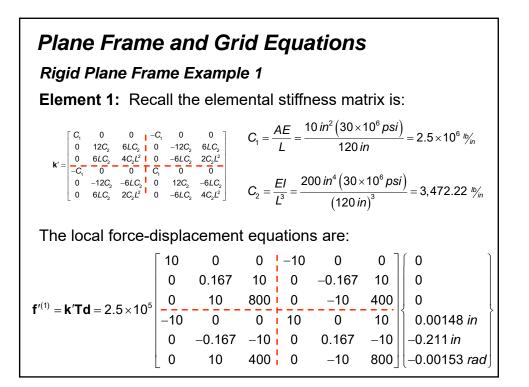


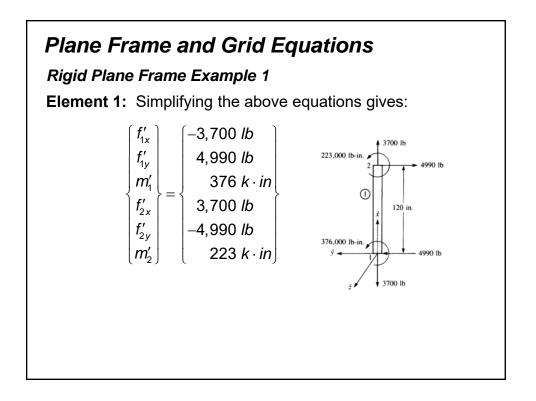






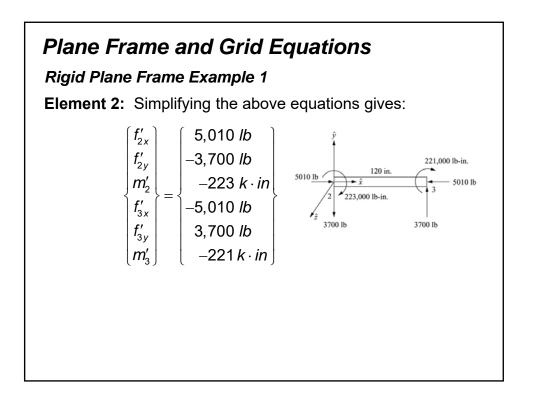
Plane Frame and Grid Equations Rigid Plane Frame Example 1											
Element 1: The element force-displacement equations can											
be obtained using $\mathbf{f'} = \mathbf{k'Td}$. Therefore, \mathbf{Td} is: $T = \begin{bmatrix} C & S & 0 & 0 & 0 & 0 \\ -S & C & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0$											
Td =	0 -1 0 0 0 0	1 0 0 0 0	0 0 1 0 0	0 0 0 -1 0	0 0 1 0	0 0 0 0 0 1	$\begin{cases} u_{1} = 0 \\ v_{1} = 0 \\ \phi_{1} = 0 \\ u_{2} = 0.211 \text{ in} \\ v_{2} = 0.00148 \text{ in} \\ \phi_{2} = -0.00153 \text{ rad} \end{cases}$		0 0 0.00148 <i>in</i> -0.211 <i>in</i> -0.00153 <i>rad</i>		



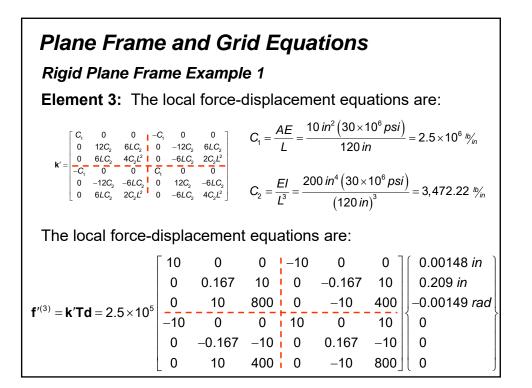


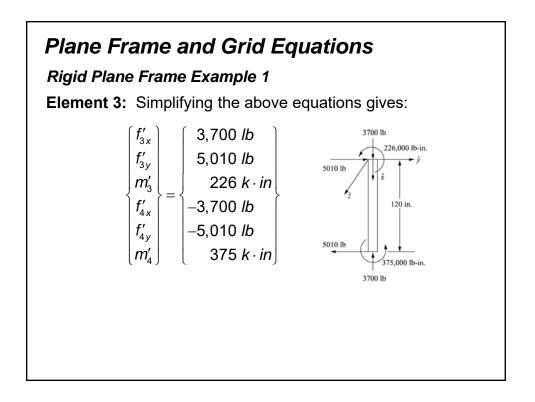
Plane Frame and Grid Equations Rigid Plane Frame Example 1											
Element 2: The element force-displacement equations can be obtained using f' = k'Td. Therefore, Td is:											
$T = \begin{bmatrix} C & S & 0 & 0 & 0 & 0 \\ -S & C & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & C & S & 0 \\ 0 & 0 & 0 & 0 & -S & C & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix} \qquad C = 1 \qquad S = 0$											
	[1	0	0	0	0	$0] (u_2 = 0.211 in) (0.211 in)]$					
	1					0 $ v_2 = 0.00148 in 0.00148 in $					
Td =	0	0	1	0	0	$\begin{array}{c c c c c c c c c c c c c c c c c c c $					
1 u =	0	0	0	1	0	$0 1 u_3 = 0.209 in 1 0.20$					
	0	0	0	0	1	$0 v_3 = -0.00148 in -0.00148 in $					
	0	0	0	0	0	$1 \left[\phi_3 = -0.00149 \ rad \right] \left[-0.00149 \ rad \right]$					

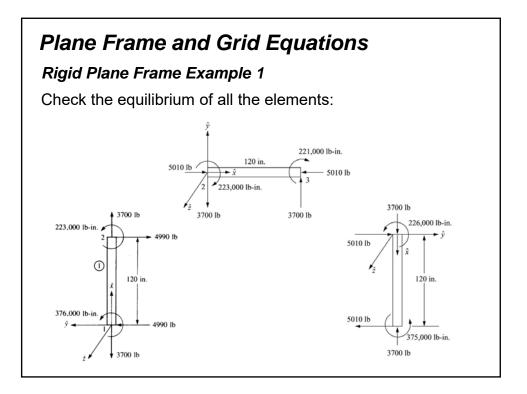
Plane Frame and Grid Equations Rigid Plane Frame Example 1										
Element 2: The local force-displacement equations are:										
$\mathbf{k}' = \begin{bmatrix} C_1 & 0 & 0 & & -C_1 & 0 & 0 \\ 0 & 12C_2 & 6LC_2 & & 0 & -12C_2 & 6LC_2 \\ 0 & -C_1 & 0 & 0 & & -C_1 & 0 & -12C_2 & 6LC_2 \\ 0 & -C_1 & 0 & 0 & -12C_2 & -6LC_2 & 1 & 0 & -6LC_2 & 2C_2L^2 \\ 0 & -12C_2 & -6LC_2 & & 0 & 12C_2 & -6LC_2 \\ 0 & -12C_2 & -6LC_2 & & 0 & 12C_2 & -6LC_2 \\ 0 & -12C_2 & -6LC_2 & 1 & 0 & 12C_2 & -6LC_2 \\ 0 & -6LC_2 & 2C_2L^2 & 0 & -6LC_2 & 4C_2L^2 \end{bmatrix} \qquad C_1 = \frac{AE}{L} = \frac{10 in^2 (30 \times 10^6 psi)}{120 in} = 2.5 \times 10^6 \frac{lb}{lm}$										
$\mathbf{k} = \begin{bmatrix} -C_1 & 0 & 0 \\ 0 & -12C_2 & -6LC_2 \\ 0 & 6LC_2 & 2C_2L^2 \end{bmatrix}$	$\begin{bmatrix} 0\\ C_2 & -6LC_2\\ C_2 & 4C_2L^2 \end{bmatrix}$	$C_{2} = \frac{EI}{L^{3}} = \frac{100 in^{4} (30 \times 10^{6} psi)}{(120 in)^{3}} = 1,736.11 \text{ b/}_{in}$								
The local force-displacement equations are:										
	∫ 10	0	0	-10	0	0]	(0.211 <i>in</i>)			
	0	0.0833	5	0	-0.0833	5	0.00148 in			
	0	5	400	0	-5	200	–0.00153 rad			
$\mathbf{t}^{(2)} = \mathbf{k}^{T}\mathbf{d} = 2.5 \times 10^{\circ}$	-10	0	0	10	0	0	0.209 in			
	0	-0.0833	-5	0	0.0833	-5	–0.00148 <i>in</i>			
$f'^{(2)} = k'Td = 2.5 \times 10^5$	0	5	200	0	-5	400	[-0.00149 <i>rad</i>]			

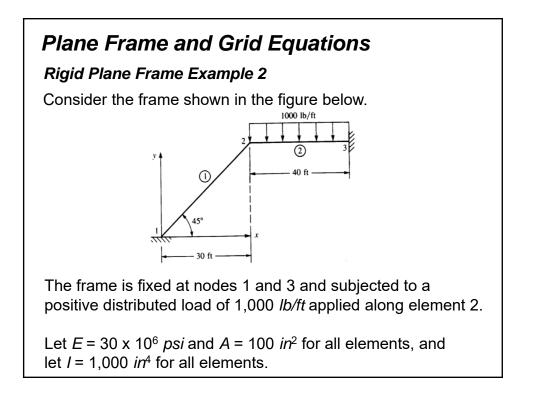


Plane Frame and Grid Equations												
Rigid Plane Frame Example 1												
Element 3: The element force-displacement equations can be obtained using f' = k'Td . Therefore, Td is:												
	$T = \begin{bmatrix} C & S & 0 & 0 & 0 & 0 \\ -S & C & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & C & S & 0 \\ 0 & 0 & 0 & -S & C & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix} \qquad C = 0 \qquad S = -1$											
Td =	0 1 0 0 0	-1 0 0 0 0 0	0 0 1 0 0	0 0 0 1 0	0 0 -1 0 0	0 0 0 0 0 1	$\begin{cases} u_3 = 0.209 \text{ in} \\ v_3 = -0.00148 \text{ in} \\ \phi_3 = -0.00149 \text{ rad} \\ u_4 = 0 \\ v_4 = 0 \\ \phi_4 = 0 \\ \phi_4 = 0 \end{cases}$		0.00148 in 0.209 in -0.00149 rad 0 0 0			







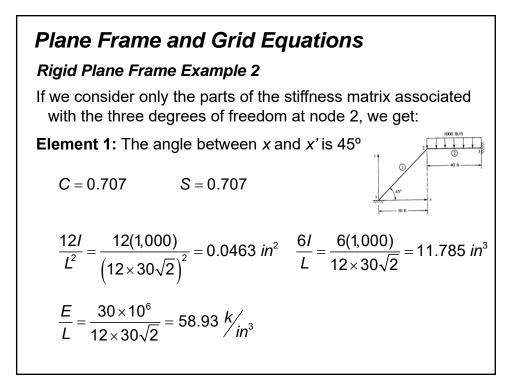


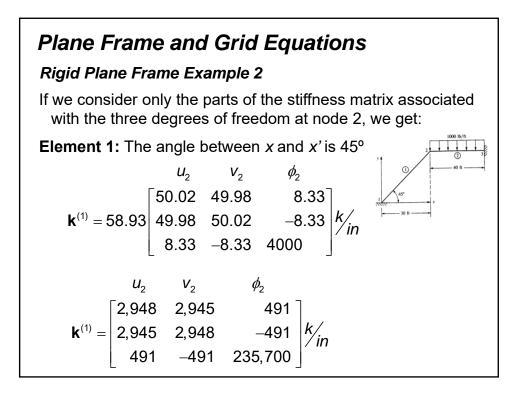
Rigid Plane Frame Example 2

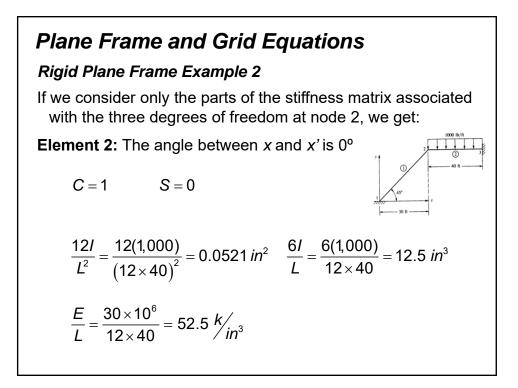
First we need to replace the distributed load with a set of equivalent nodal forces and moments acting at nodes 2 and 3.

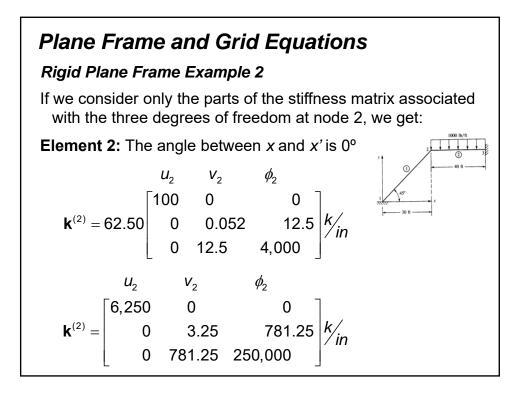
For a beam with both end fixed, subjected to a uniform distributed load, *w*, the nodal forces and moments are:

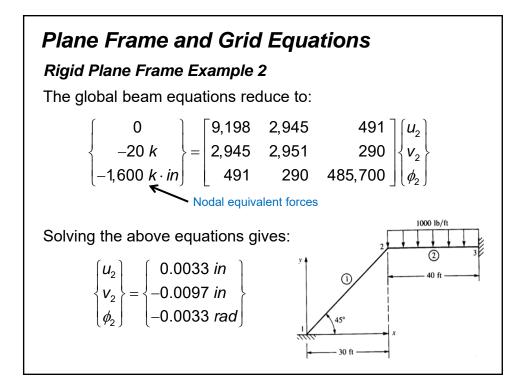
$$f_{2y} = f_{3y} = -\frac{wL}{2} = -\frac{(1,000 \, lb \, / \, ft)40 \, ft}{2} = -20k$$
$$m_2 = -m_3 = -\frac{wL^2}{12} = -\frac{(1,000 \, lb \, / \, ft)(40 \, ft)^2}{12} = -133,333 \, lb \cdot ft$$
$$= 1,600 \, k \cdot in$$

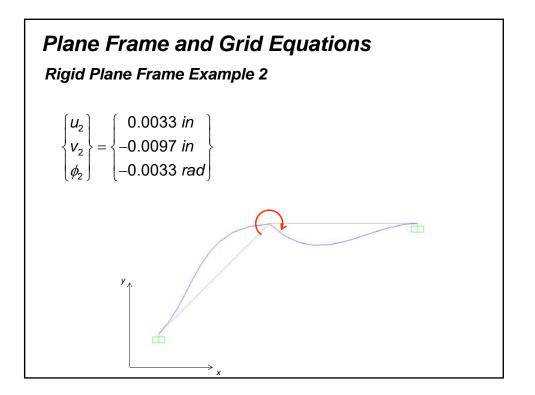


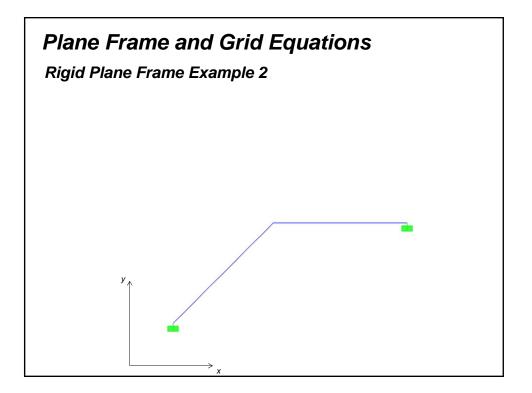


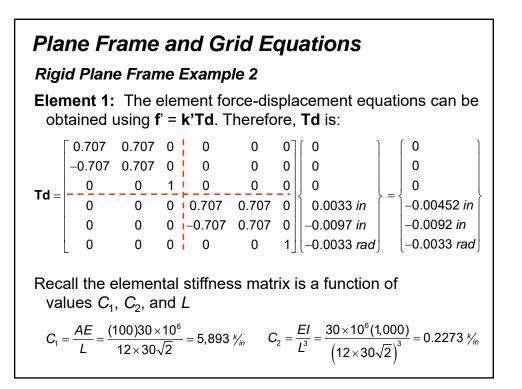


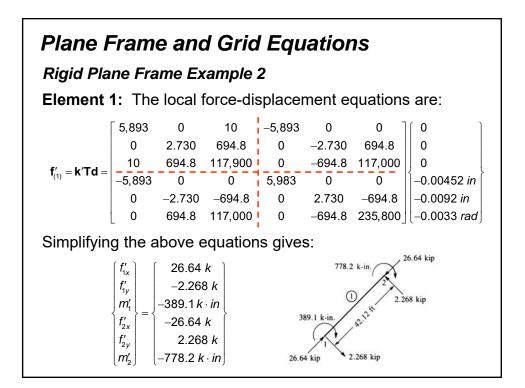


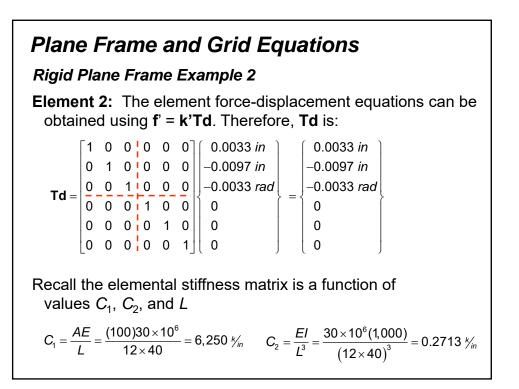


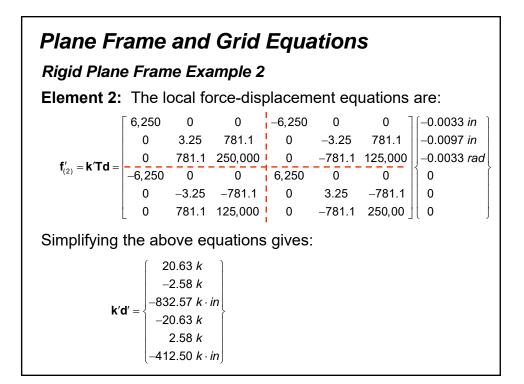


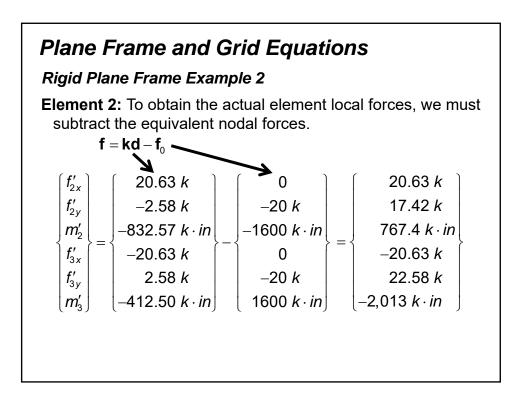


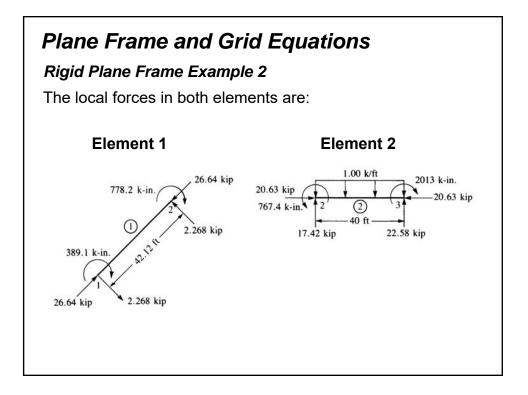






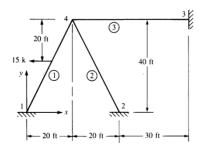




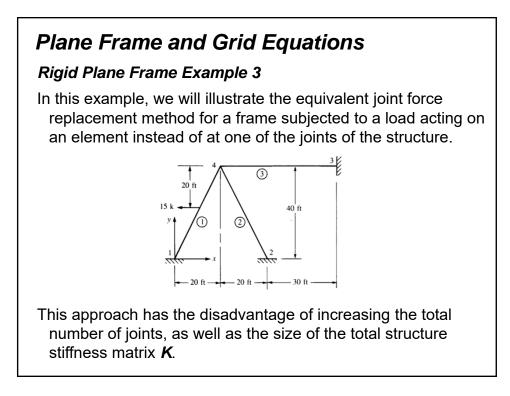


Rigid Plane Frame Example 3

In this example, we will illustrate the equivalent joint force replacement method for a frame subjected to a load acting on an element instead of at one of the joints of the structure.



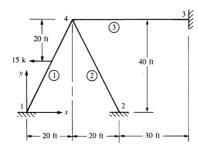
Since no distributed loads are present, the point of application of the concentrated load could be treated as an extra joint in the analysis.



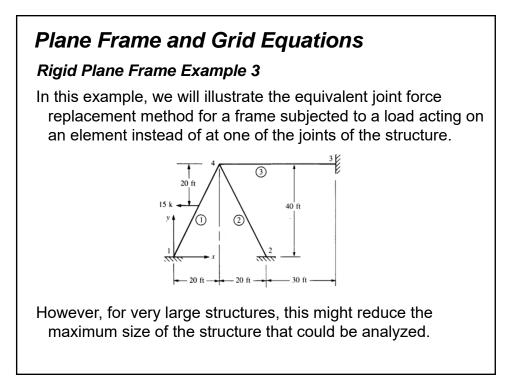


Rigid Plane Frame Example 3

In this example, we will illustrate the equivalent joint force replacement method for a frame subjected to a load acting on an element instead of at one of the joints of the structure.

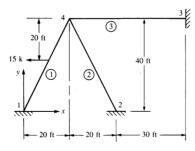


For small structures solved by computer, this does not pose a problem.

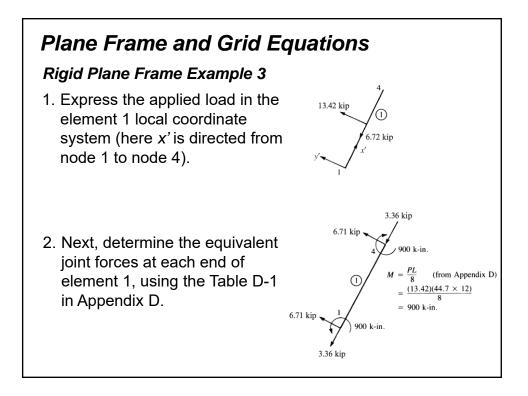


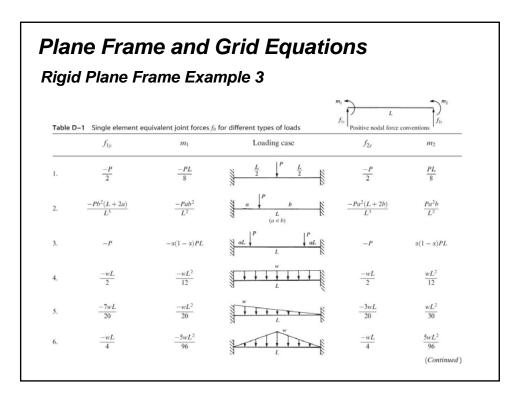
Rigid Plane Frame Example 3

In this example, we will illustrate the equivalent joint force replacement method for a frame subjected to a load acting on an element instead of at one of the joints of the structure.

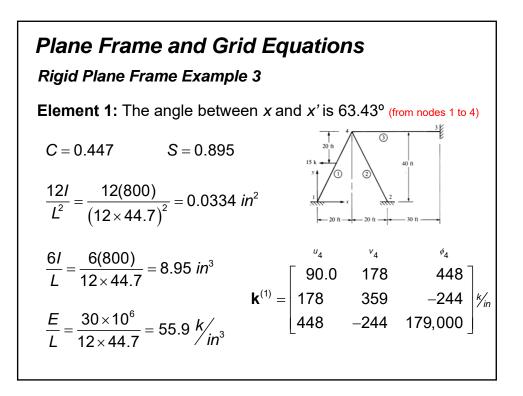


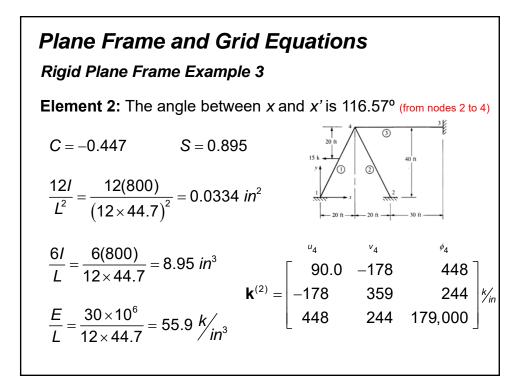
The frame is fixed at nodes 1, 2, and 3 and subjected to a concentrated load of 15 *k* applied at mid-length of element 1. Let $E = 30 \times 10^6 psi$, $A = 8 in^2$, and let $I = 800 in^4$ for all elements.

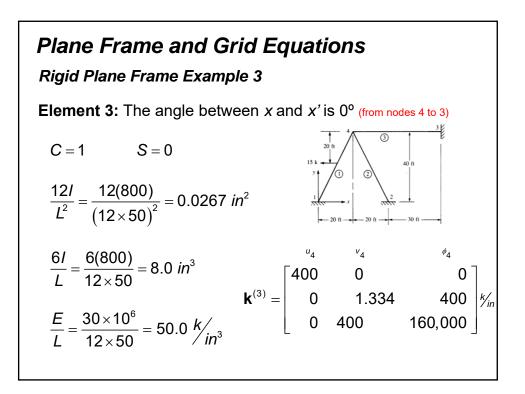




Plane Frame and Grid Equations Rigid Plane Frame Example 3 Then transform the equivalent joint forces from the local coordinate system forces into the global coordinate system forces, using the equation: f = T^Tf' These global joint forces are: 4. Then we analyze the structure, using the equivalent joint forces (plus actual joint forces, if any) in the usual manner. 5. The final internal forces developed at the ends of each element may be obtained by subtracting Step 2 joint forces from Step 4 joint forces.







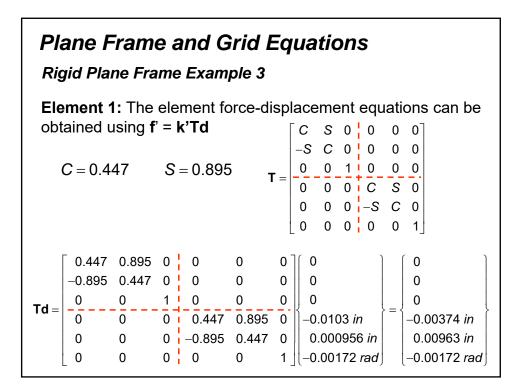
Rigid Plane Frame Example 3

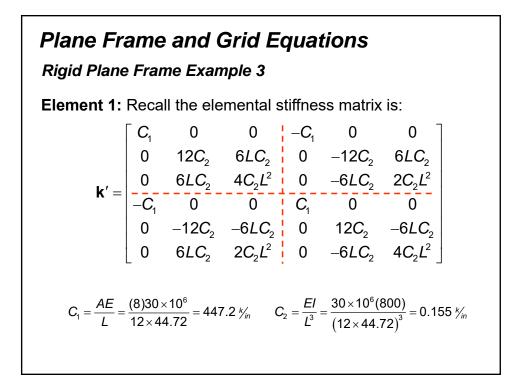
The global beam equations reduce to:

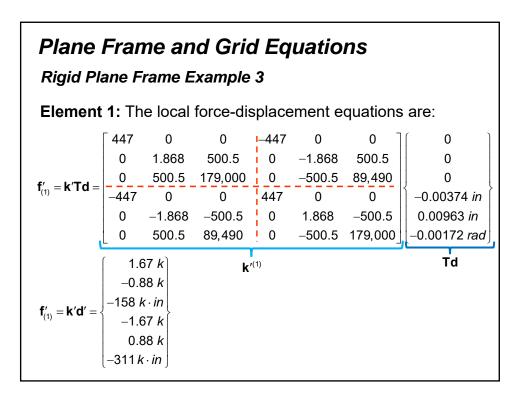
$\begin{cases} -7.5 \ k \\ 0 \\ -900 \ k \cdot in \end{cases}$		582	0	896	$\left \left(u_{4} \right) \right $
0	=	0	719	400	$\left \left\{ V_{4}\right\} \right $
900 <i>k</i> · in		896	400	518,000	$\left[\phi{4}\right]$

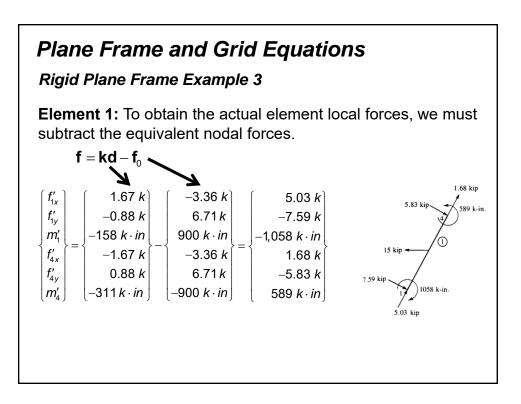
Solving the above equations gives:

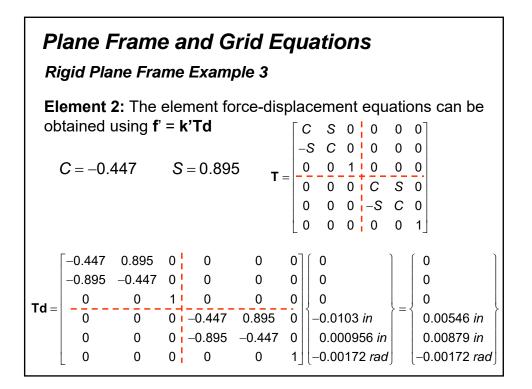
$$\begin{cases} u_4 \\ v_4 \\ \phi_4 \end{cases} = \begin{cases} -0.0103 \text{ in} \\ 0.000956 \text{ in} \\ -0.00172 \text{ rad} \end{cases}$$

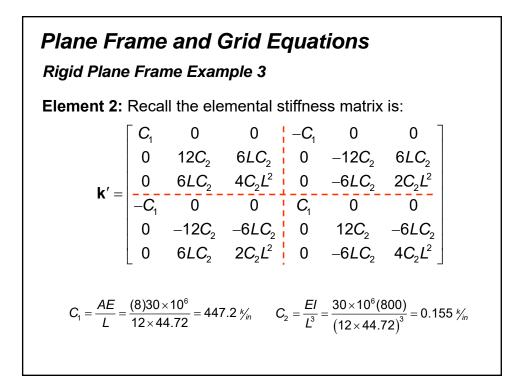


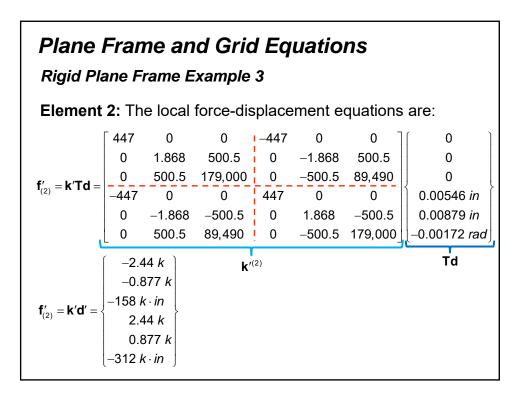








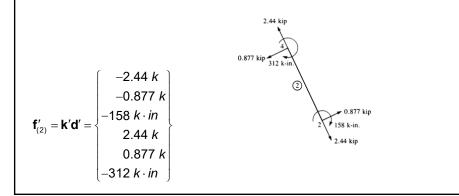


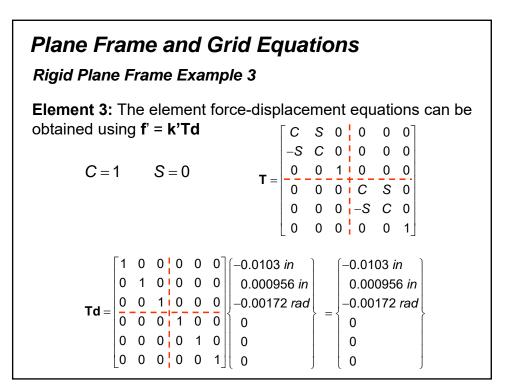


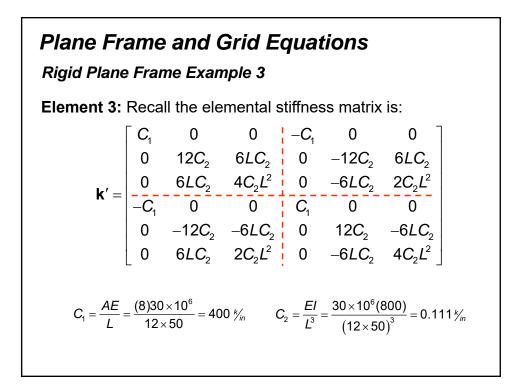
Rigid Plane Frame Example 3

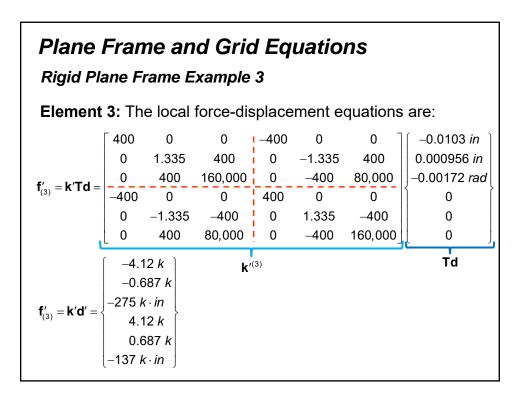
Element 2: Since there are no applied loads on element 2, there are no equivalent nodal forces to account for.

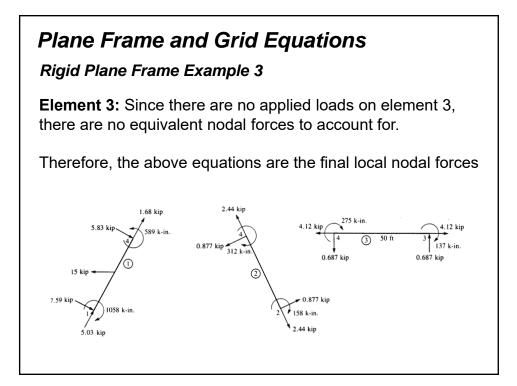
Therefore, the above equations are the final local nodal forces

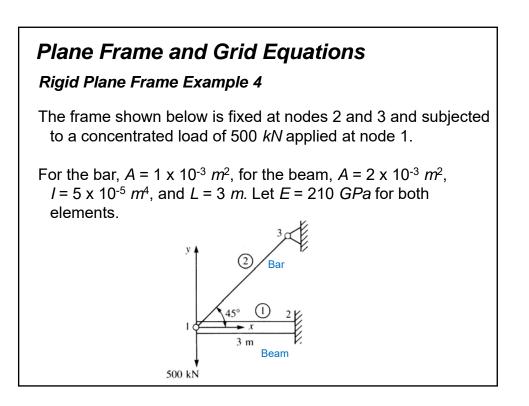


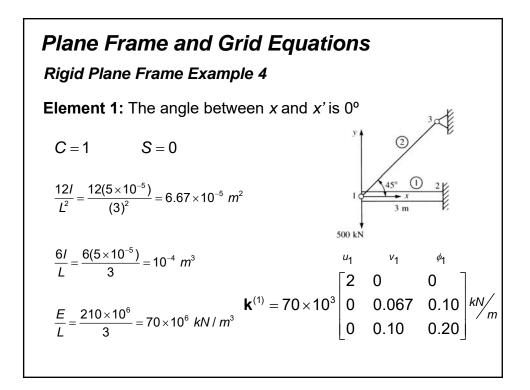


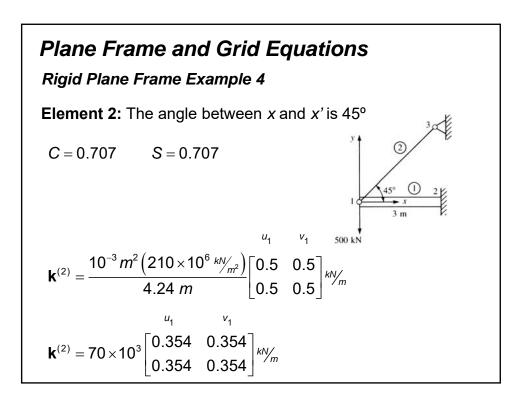


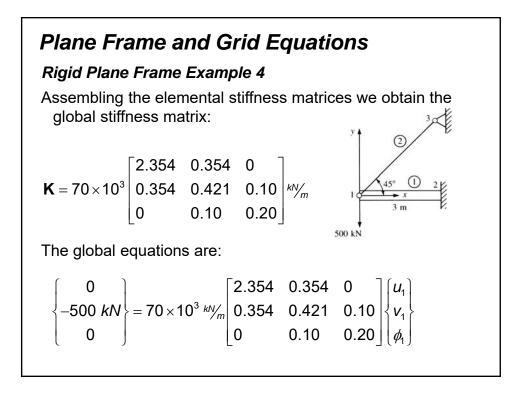


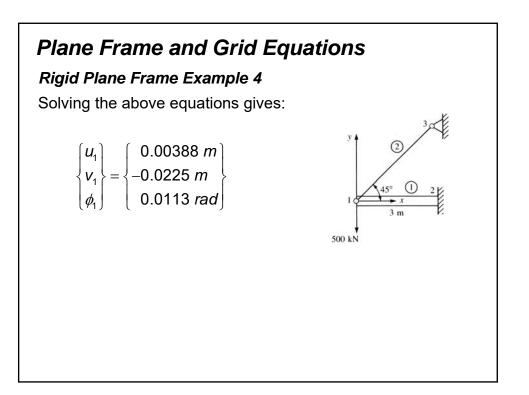


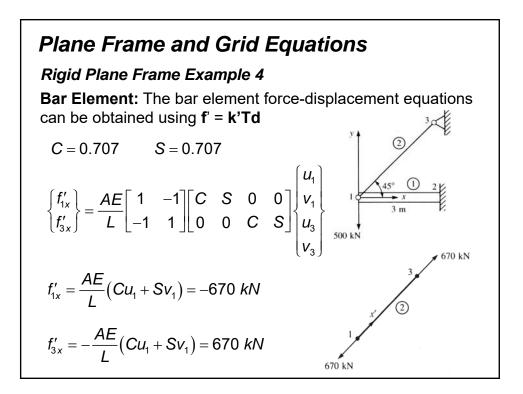


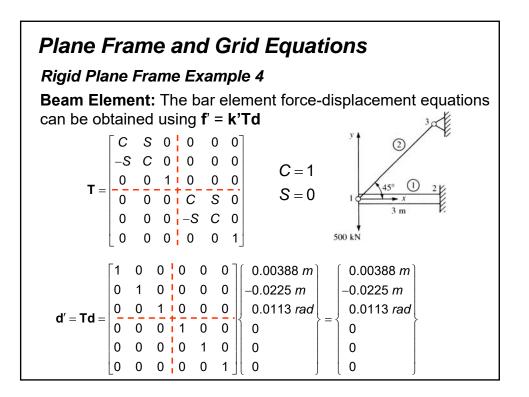


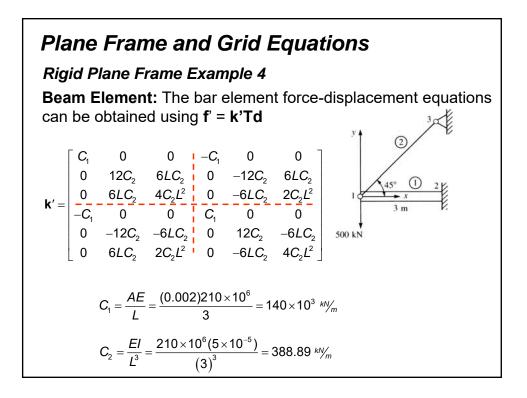


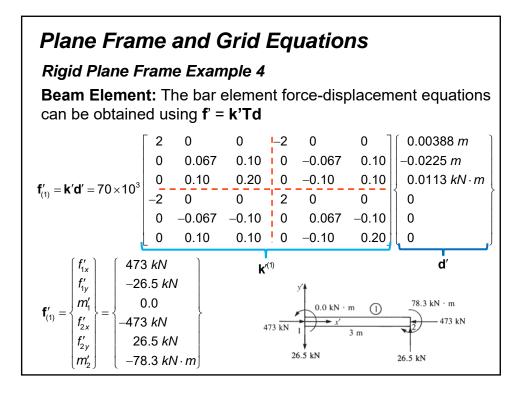


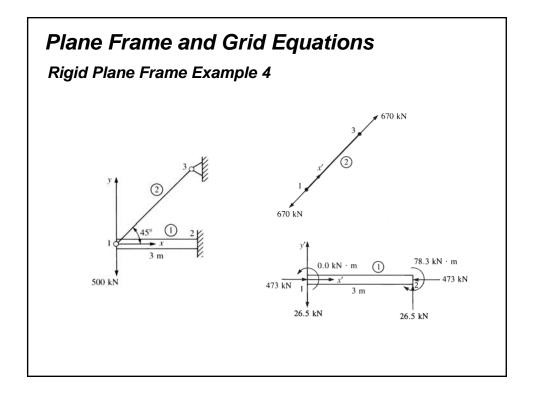










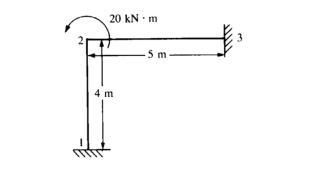


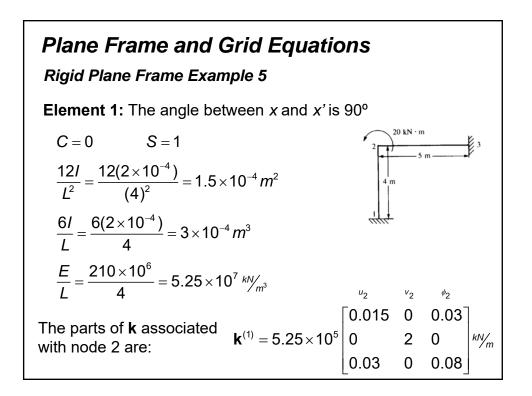


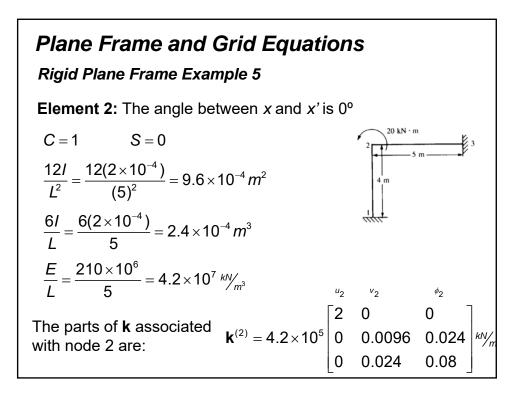
Rigid Plane Frame Example 5

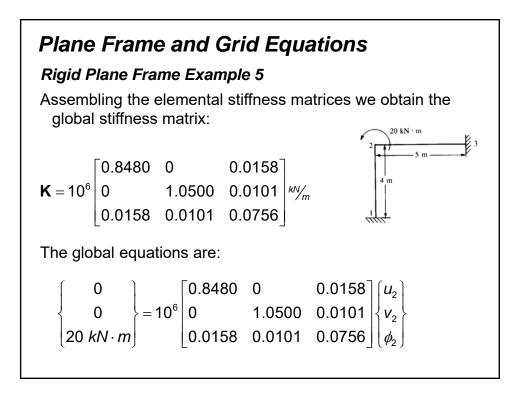
The frame is fixed at nodes 1 and 3 and subjected to a moment of 20 *kN-m* applied at node 2

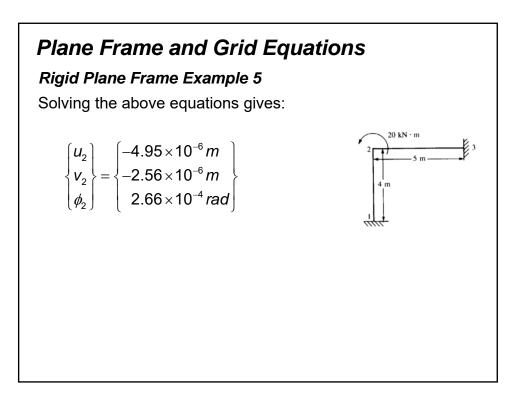
Assume $A = 2 \times 10^{-2} m^2$, $I = 2 \times 10^{-4} m^4$, and E = 210 GPa for all elements.

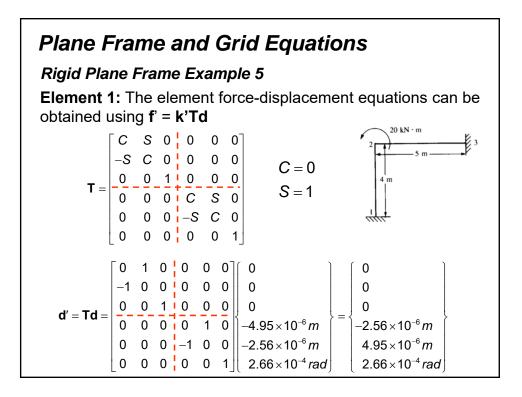


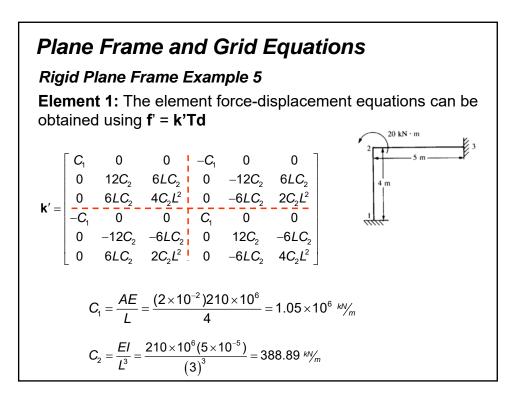




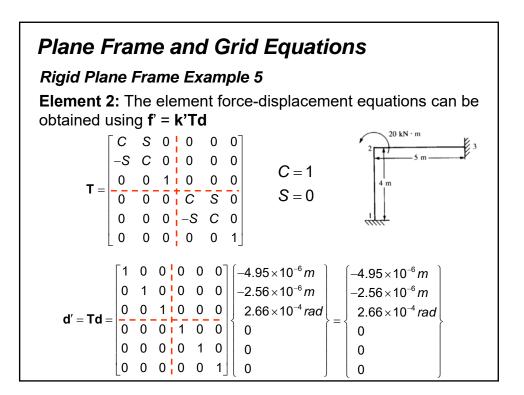


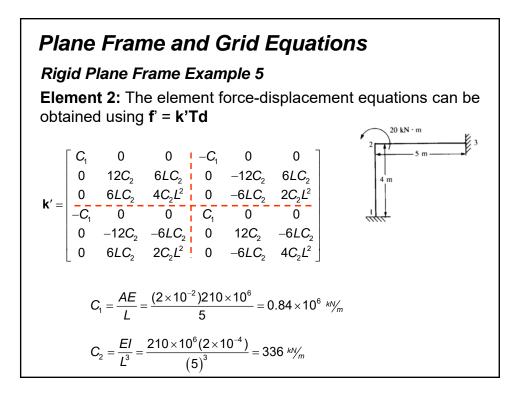


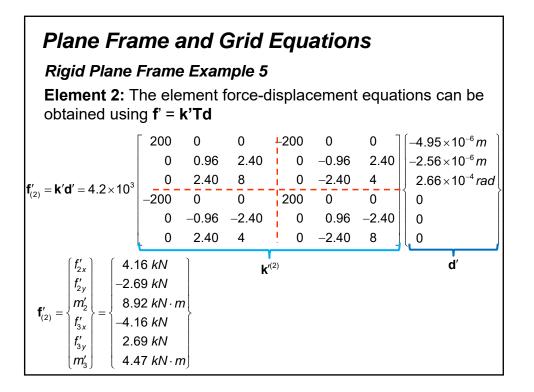




Plane Frame and Grid Equations Rigid Plane Frame Example 5 Element 1: The element force-displacement equations can be obtained using **f**' = **k'Td** 0 –200 0 200 0 0 | 0 0 1.5 3 0 -1.5 3 0 $\mathbf{f}'_{(1)} = \mathbf{k}'\mathbf{d}' = 5.25 \times 10^3 \begin{vmatrix} 0 & 3 & 8 & 0 & -3 & 4 \\ -200 & 0 & 0 & 200 & 0 & 0 \end{vmatrix} \begin{vmatrix} 0 & -3 & 4 & 0 \\ -2.56 \times 10^{-6} & m \end{vmatrix}$ 0 -1.5 -3 | 0 1.5 -3 || $4.95 \times 10^{-6} m$ 0 3 4 0 -3 8 2.66 $\times 10^{-4}$ rad 2.69 *kN* ď f_{1x}' **K**⁽¹⁾ $\mathbf{f}_{(1)}' = \begin{cases} f_{1y}' \\ f_{1y}' \\ m_1' \\ f_{2x}' \\ \end{cases} = \begin{cases} 2.00 \text{ kN} \\ 4.2 \text{ kN} \\ 5.59 \text{ kN} \cdot m \\ -2.69 \text{ kN} \end{cases}$ f'_2y -4.2 kN $|m'_{2}|$ 11.17 kN · m





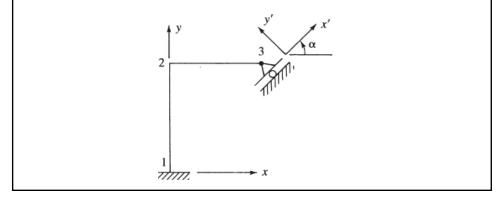


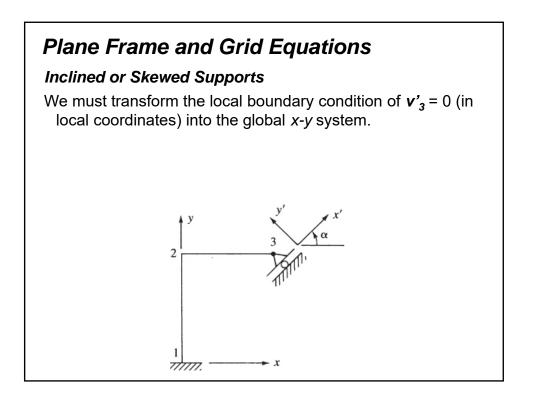
Plane Frame and Grid Equations

Inclined or Skewed Supports

If a support is inclined, or skewed, at some angle α for the global *x* axis, as shown below.

The boundary conditions on the displacements are not in the global x-y directions but in the x'-y' directions.





Plane Frame and Grid Equations

Inclined or Skewed Supports

Therefore, the relationship between of the components of the displacement in the local and the global coordinate systems at node 3 is:

$\left[U'_{3} \right]$	$\int \cos \alpha$	$\sin lpha$	0	$\left[u_{3} \right]$
$\left\{ V'_{3} \right\} =$	$-\sin \alpha$	$\cos \alpha$	0	$\{V_3\}$
$\left \phi'_{3}\right $	0	0		$\left[\phi_{3}\right]$

We can rewrite the above expression as:

		$\cos \alpha$	$\sin lpha$	0
$\{d'_3\} = [t_3]\{d_3\}$	$\begin{bmatrix} t_3 \end{bmatrix} =$	$-\sin lpha$	$\cos \alpha$	0
		$\cos \alpha$ -sin α	0	1
		-		_

Plane Frame and Grid Equations

Inclined or Skewed Supports

We can apply this sort of transformation to the entire displacement vector as:

 $\{d'\} = [T_i]\{d\}$ or $\{d\} = [T_i]^T\{d'\}$

where the matrix [*T_i*] is:

	[/]	[0]	[0]
$[T_i] =$	[0]	[/]	[0] [0]
	[0]	[0]	$[t_3]$

Both the identity matrix [/] and the matrix $[t_3]$ are 3 x 3 matrices.

Plane Frame and Grid Equations

Inclined or Skewed Supports

The force vector can be transformed by using the same transformation. $\{f'\} = [T_i]\{f\}$

In global coordinates, the force-displacement equations are: $\{f\} = [K]\{d\}$

Applying the skewed support transformation to both sides of the force-displacement equation gives:

 $[T_i]{f} = [T_i][K]{d}$

By using the relationship between the local and the global displacements, the force-displacement equations become:

 $[T_i]{f} = [T_i][K][T_i]^T {d'} \implies {f'} = [T_i][K][T_i]^T {d'}$

