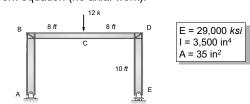
- Applying the virtual work equations to a frame structure is as simple as separating the frame into a series of "beams" and summing the virtual work for each section.
- In addition, when evaluating the deformation of a frame structure, you may have to consider both bending and axial internal force components.

**Virtual Work for Frames** 

Compute the *deflection* at point C on the frame shown below.

Include only the effects of bending in your virtual work equation (no axial work).



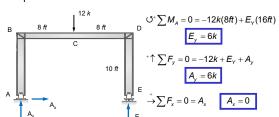
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1

# **Virtual Work for Frames**

The first step is to find the equation for moment in each section of the frame due to the *real loads*.

To do develop the moment expression we need the reaction a points A and E.



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# **Virtual Work for Frames**

The next step is to find the equation for moment in each section of the frame.

Consider section AB

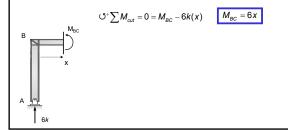
$$O^+\sum M_{cut} = 0 = M_{AB} \qquad M_{AB} = 0$$



/

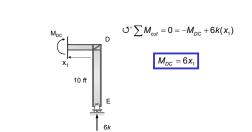
# Virtual Work for Frames

Consider section BC



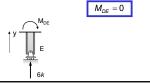
Virtual Work for Frames

Consider section DC



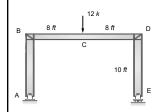
Consider section ED

$$O^+ \sum M_{cut} = 0 = -M_{DE}$$



#### **Virtual Work for Frames**

In this problem, the virtual moments are the real moments divided by 12 (from superposition).



Section	М	m
AB	0	0
ВС	6x	0.5x
DC	6x <sub>1</sub>	0.5x <sub>1</sub>
DE	0	0

# **Virtual Work for Frames**

The virtual work equations are:

$$\Delta_{C} = \int\limits_{A}^{B} \frac{M_{AB}m_{AB}}{EI} dy + \int\limits_{B}^{C} \frac{M_{BC}m_{BC}}{EI} dx + \int\limits_{D}^{C} \frac{M_{DC}m_{DC}}{EI} dx + \int\limits_{D}^{E} \frac{M_{DE}m_{DE}}{EI} dy$$

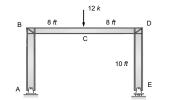
Substituting the moment expressions into the virtual work equation and integrating yields the following:

$$\begin{split} \Delta_{\text{C}} &= \int_{0}^{8} \frac{(6x)x}{2EI} dx + \int_{0}^{8} \frac{(6x_{1})x_{1}}{2EI} dx_{1} &= \int_{0}^{8} \frac{6x^{2}}{EI} dx = \frac{2x^{3}}{EI} \bigg|_{0}^{8} \\ &= \frac{1,024 \text{ kft}^{3}}{EI} &= \frac{1,024 \text{ kft}^{3} (1,728 \text{in}^{3} / \text{ft}^{3})}{(29,000 \text{ksi})(3,500 \text{in}^{4})} &= \boxed{0.017 \text{in}} \end{split}$$

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# **Virtual Work for Frames**

Compute the *slope* at point C on the frame shown below. Include only the effects of bending in your virtual work equation (no axial work).



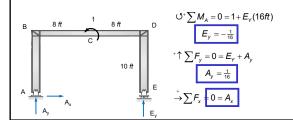
E = 29,000 psi  $I = 3,500 in^4$  $A = 35 \text{ in}^2$ 

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# **Virtual Work for Frames**

Find the moments in the frame due to a virtual couple.

First, find the reaction in the frame to the virtual couple.



**Virtual Work for Frames** 

The next step is to find the equation for moment in each section of the frame.

Consider section AB

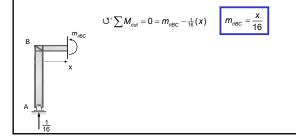
 $O^{+}\sum M_{cut} = 0 = m_{\theta AB} \qquad m_{\theta AB} = 0$ 



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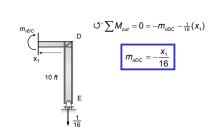
Consider section BC



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# **Virtual Work for Frames**

Consider section DC



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# **Virtual Work for Frames**

Consider section ED

$$O^{+}\sum M_{cut} = 0 = -m_{\theta DE}$$

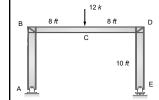




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# **Virtual Work for Frames**

The following table lists the moment expression due to the real loading and the moment expression due to a virtual couple at point C



М	$m_{\scriptscriptstyle{\theta}}$
0	0
6x	x/16
6x <sub>1</sub>	-x <sub>1</sub> /16
0	0
	0 6x 6x <sub>1</sub>

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# **Virtual Work for Frames**

The virtual work equations are:

$$\theta_{c} = \int\limits_{A}^{B} \frac{M_{AB} m_{\partial AB}}{EI} dy + \int\limits_{B}^{c} \frac{M_{BC} m_{\partial BC}}{EI} dx + \int\limits_{D}^{c} \frac{M_{DC} m_{\partial DC}}{EI} dx + \int\limits_{E}^{D} \frac{M_{DE} m_{\partial DE}}{EI} dy$$

Substituting the moment expressions into the virtual work equation and integrating yields the following:

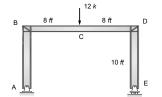
$$\theta_{\rm C} = \int_{0}^{8} \frac{(6x)x}{16EI} dx - \int_{0}^{8} \frac{(6x_1)x_1}{16EI} dx_1 = 0$$

The slope at point C is zero

# **Virtual Work for Frames**

Repeat the previous example and include the effects of axial

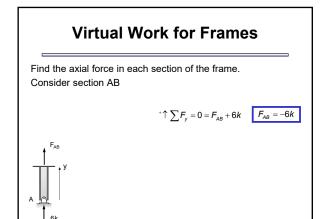
In order to compute the axial work, we need the axial force in the real and virtual loading systems



E = 29,000 *psi* I = 3,500 in<sup>4</sup>  $A = 35 \text{ in}^2$ 

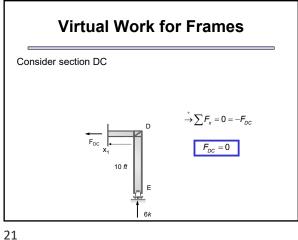
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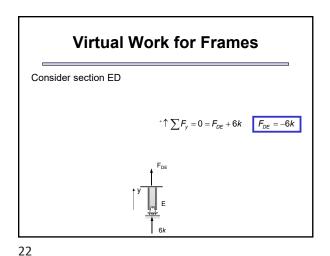
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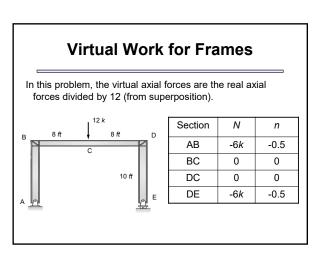


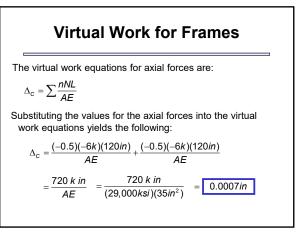
**Virtual Work for Frames** Consider section BC  $\stackrel{^{+}}{\rightarrow} \sum F_{x} = 0 = F_{BC}$ 

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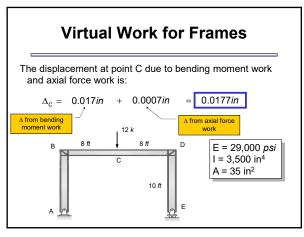


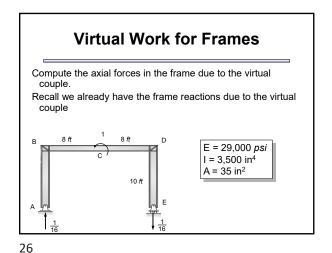




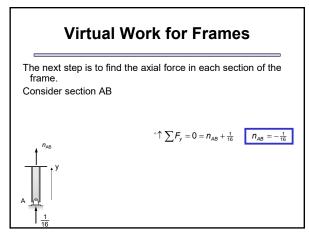


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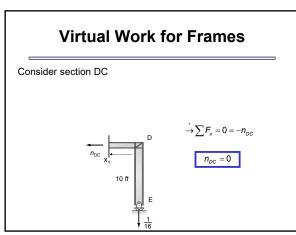


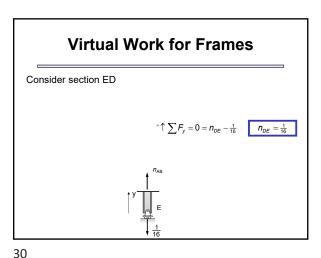


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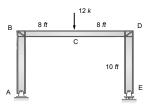


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The real axial forces and the virtual axial forces due to a unit virtual couple are:



Section	N	$n_{\theta}$
AB	-6 <i>k</i>	-1/16
BC	0	0
DC	0	0
DE	-6 <i>k</i>	1/16

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# Virtual Work for Frames

The virtual work equations for axial forces are:

$$\theta_{\rm C} = \sum \frac{n_{\theta} NL}{AE}$$

Substituting the values for the axial forces into the virtual work equations yields the following:

$$\theta_{\rm C} = \frac{(6k)(120in)}{16AE} - \frac{(6k)(120in)}{16AE} = 0$$

The contribution to the slope at point C from the axial energy is slope is zero.

The total slope at point C due to bending moment and axial force work is zero.

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#### **Virtual Work for Frames**

- In problems involving both bending and axial deformation, be careful with the units.
- Also, note that the contribution of the axial deformation is 5% of the total deformation.
- > This is typical of the relative size of the bending and the axial effects in frame-deflection problems.
- Therefore, it is usually permissible to neglect the effect of axial deformation in such cases.

#### **Virtual Work for Frames**

- > The primary cause of deformation in beams and frames is due to **bending strain**.
- However, in some structures, the additional deformation due to axial and shear forces, torsion, and perhaps temperature may be important.
- We have already discussed deformation due to bending moments and axial forces.
- In this section, we will consider the effect of shear, torsion, and temperature on the deformation of linear elastic structures.

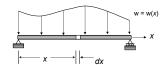
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#### **Virtual Work for Frames**

#### Virtual Strain Energy From Shear

Consider the following beam and a small element dx





# Virtual Strain Energy From Shear

The shearing deformation dy caused by the real loads is dy = γdx, where γ is the shear strain.

Virtual Work for Frames

- Since we are assuming the material is linear and elastic, then Hooke's law applies
- The shear strain is related to the shear stress by  $\gamma = \tau/G$ , where  $\tau$  is the shear stress and G is the shearing modulus of elasticity.



#### **Virtual Strain Energy From Shear**

- The shear stress may be calculated as τ = K(V/A)dx, where K is a form factor that depends of the shape of the beam's cross-sectional area A.
- Combining these two expressions gives dy = KV/(GA) dx.
- The internal virtual work done by the virtual shear force v acting on the beam before it is deformed by the real loads is dU<sub>i</sub> = v dy



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### **Virtual Work for Frames**

#### Virtual Strain Energy From Shear

Integrating the expression dU<sub>i</sub> = v dy over the entire beam gives:

$$U_{shear} = \int_{0}^{L} K\left(\frac{vV}{GA}\right) dx$$

Remember that *v* is the shear due to the virtual load and *V* is the shear due to the real loads.

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# **Virtual Work for Frames**

#### Virtual Strain Energy From Shear

➤ Integrating the expression dU<sub>i</sub> = v dy over the entire beam gives:

$$U_{\text{shear}} = \int_{0}^{L} K\left(\frac{vV}{GA}\right) dx$$

shear ∫ (GA)

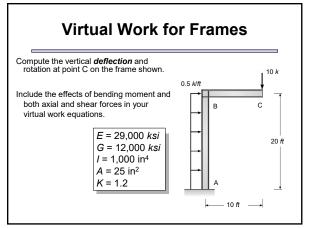
➤ The form factor K is based on the

cross-sectional area: **K** = 1.2 for rectangular sections

K = 10/9 for circular sections

K≈ 1 for I-beams, where A is the area of the web

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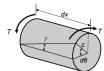


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#### **Virtual Work for Frames**

#### **Virtual Strain Energy From Torsion**

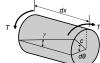
- For example, consider a circular cross-section where no wrapping of the section occurs.
- For non-circular sections a more rigorous analysis is required.



#### Virtual Work for Frames

# Virtual Strain Energy From Torsion

- This torque causes a shear strain:  $\gamma = \frac{c d\theta}{dx}$
- For a linear-elastic response:  $\gamma = \frac{\tau}{G}$

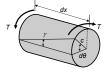


$$\tau = \frac{Tc}{J}$$

#### **Virtual Strain Energy From Torsion**

The angle of twist:

$$d\theta = \frac{\gamma}{c}dx = \frac{\tau}{Gc}dx = \frac{T}{GJ}dx$$

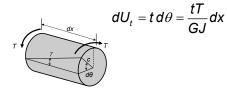


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#### **Virtual Work for Frames**

#### **Virtual Strain Energy From Torsion**

If a virtual load is applied to the structure that causes an internal virtual torque *t*, then after applying the real loads, virtual strain energy will be:



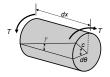
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# **Virtual Work for Frames**

#### **Virtual Strain Energy From Torsion**

Integrating the virtual strain over the length of the member yields:

 $U_t = \frac{tTL}{GJ}$ 



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#### **Virtual Work for Frames**

#### **Virtual Strain Energy From Temperature**

- Consider a structure member is subjected to a temperature difference across its depth.
- For discussion, we will choose the most common case of a beam having a neutral axis located at the mid-depth c of the beam
- First compute the amount of rotation of a differential element dx of the beam caused by the thermal gradient.

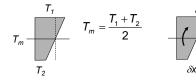
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#### **Virtual Work for Frames**

#### **Virtual Strain Energy From Temperature**



$$\delta \mathbf{x} = \alpha \, \Delta T_m \, \mathbf{d} \mathbf{x}$$



#### Virtual Work for Frames

#### Virtual Strain Energy From Temperature

If a virtual load is applied to the structure that causes an internal virtual torque m, then after applying the real loads, virtual strain energy will be:

$$dU_{temp} = \int_{0}^{L} \frac{m\alpha \, \Delta T_{m}}{c} dx$$

- Unless otherwise stated, in this course we will consider only beam and frame deflections due to bending.
- The additional deflections caused by the shear and axial force alter the deflections by only a few percent and are generally ignored for even "small" two- and three-member frames of one-story height.

# Any questions?